## $\Delta$

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\author{

* Chapter 1: Introduction to Statics <br> * Chapter 2: Force Systems <br> Chapter 3: Equilibrium <br> - Chapter 4: Structures <br> Chapter 5: Distributed Forces <br> - Chapter 6: Friction
}
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### 2.2 FORCE

Properties of a single force:*Action of one body on another

* Action which tends to cause acceleration
*Vector quantity (Magnitude and Direction)
*Forces may be combined by vector addition


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### 2.2 FORCE

$\square$ Complete specification of the action of this force must include:

* Magnitude
* Direction

Point of application
$\checkmark$ We must treat it as a fixed vectorExternal and Internal Effects

- External Forces:
$\checkmark$ Applied forces

$\checkmark$ Reactive forces
: External forces lead to creation of internal forces
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### 2.2 FORCE

## $\square$ Principle of Transmissibility

* When dealing with the mechanics of a rigid body, we ignore deformations in the body
* The external effects of the exerted force should be same
- So it is not necessary to restrict the action of an applied force to a given point
* For example:
$\checkmark$ Force P may be applied at A or at B or at any other point on its line of action
$\checkmark$ External effects: bearing support at O and roller support at C

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### 2.2 FORCE

$\square$ Principle of Transmissibility:

* A force may be applied at any point on its given line of action without altering the resultant effects of the force external to the rigid body on which it acts.
:The force may be treated as a sliding vector:
$\checkmark$ Magnitude
$\checkmark$ Direction
$\checkmark$ Line of action


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### 2.2 FORCE

Force ClassificationContact or Body forces:
$\checkmark$ A contact force is produced by direct physical contact
$\checkmark$ A body force is generated by virtue of the position of a body within a force field (such as a gravitational)

Cable tension $P$

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### 2.2 FORCE

## Force Classification

* Concentrated or Distributed forces
$\checkmark$ Actually, almost all forces are distributed forces.
$\checkmark$ When the dimensions of the area are very small compared with the other dimensions of the body, we may consider the force to be concentrated

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### 2.2 FORCE

- Action and Reaction

According to Newton's third law, the action of a force is always accompanied by an equal and opposite reaction

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### 2.2 FORCE

$\square$ Concurrent Forces

* Their lines of action intersect at that point
:They can be added using the parallelogram law in their common plane

$$
\mathbf{R}=\mathbf{F}_{1}+\mathbf{F}_{2}
$$


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### 2.2 FORCE

$\square$ Vector Components

* We often need to replace a force by its vector components in directions which are convenient for a given application

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### 2.2 FORCE

$\square$ A Special Case of Vector Addition

* Parallel Forces
$\checkmark$ Finding correct line of action



### 2.3 Rectangular Components

$\square$ The most common two-dimensional resolution of a force vector: Rectangular Components


$$
\begin{gathered}
\mathbf{F}=\mathbf{F}_{x}+\mathbf{F}_{y} \\
\mathbf{F}=F_{x} \mathbf{i}+F_{y} \mathbf{j} \\
F_{x}=F \cos \theta \quad F=\sqrt{F_{x}^{2}+F_{y}^{2}}=|\mathbf{F}| \\
F_{y}=F \sin \theta \quad \theta=\tan ^{-1} \frac{F_{y}}{F_{x}}
\end{gathered}
$$

### 2.3 Rectangular Components

$\square$ Determining the Components of a Force


$$
\begin{array}{ll}
F_{x}=F \sin \beta & F_{x}=-F \cos \beta \\
F_{y}=F \cos \beta & F_{y}=-F \sin \beta
\end{array}
$$




$$
\begin{aligned}
& F_{x}=F \sin (\pi-\beta) \\
& F_{y}=-F \cos (\pi-\beta)
\end{aligned}
$$


$F_{x}=F \cos (\beta-\alpha)$
$F_{y}=F \sin (\beta-\alpha)$

### 2.3 Rectangular Components

$\square$ Finding the sum or resultant R of two forces (which are concurrent)

* Summing each component separately

$$
\mathbf{R}=\mathbf{F}_{1}+\mathbf{F}_{2}=\left(F_{1_{x}} \mathbf{i}+F_{1_{y}} \mathbf{j}\right)+\left(F_{2_{x}} \mathbf{i}+F_{2_{y}} \mathbf{j}\right)
$$

$$
R_{x} \mathbf{i}+R_{y} \mathbf{j}=\left(F_{1_{x}}+F_{2_{x}}\right) \mathbf{i}+\left(F_{1_{y}}+F_{2_{y}}\right) \mathbf{j}
$$

$$
\begin{aligned}
& R_{x}=F_{1_{x}}+F_{2_{x}}=\Sigma F_{x} \\
& R_{y}=F_{1_{y}}+F_{2_{y}}=\Sigma F_{y}
\end{aligned}
$$



## Sample Problem 2/1

The forces $\mathbf{F}_{1}, \mathbf{F}_{2}$, and $\mathbf{F}_{3}$, all of which act on point $A$ of the bracket, are specified in three different ways. Determine the $x$ and $y$ scalar components of each of the three forces.


$$
\begin{aligned}
& F_{1_{x}}=600 \cos 35^{\circ}=491 \mathrm{~N} \\
& F_{1_{y}}=600 \sin 35^{\circ}=344 \mathrm{~N}
\end{aligned}
$$

$$
\begin{aligned}
& F_{2_{x}}=-500\left(\frac{4}{5}\right)=-400 \mathrm{~N} \\
& F_{2_{y}}=500\left(\frac{3}{5}\right)=300 \mathrm{~N}
\end{aligned}
$$

$$
\alpha=\tan ^{-1}\left[\frac{0.2}{0.4}\right]=26.6^{\circ}
$$

$$
F_{3_{x}}=F_{3} \sin \alpha=800 \sin 26.6^{\circ}=358 \mathrm{~N}
$$

$$
F_{3_{y}}=-F_{3} \cos \alpha=-800 \cos 26.6^{\circ}=-716 \mathrm{~N}
$$

$$
\begin{aligned}
\mathbf{F}_{3}=F_{3} \mathbf{n}_{A B}=F_{3} \frac{\overrightarrow{A B}}{\overrightarrow{A B}} & =800\left[\frac{0.2 \mathbf{i}-0.4 \mathbf{j}}{\sqrt{(0.2)^{2}+(-0.4)^{2}}}\right] \\
& =800[0.447 \mathbf{i}-0.894 \mathbf{j}] \\
& =358 \mathbf{i}-716 \mathbf{j} \mathrm{~N}
\end{aligned}
$$

$$
\begin{aligned}
& F_{3_{x}}=358 \mathrm{~N} \\
& F_{3 y}=-716 \mathrm{~N}
\end{aligned}
$$

## Sample Problem 2/4

Forces $\mathbf{F}_{1}$ and $\mathbf{F}_{2}$ act on the bracket as shown. Determine the projection $F_{b}$ of their resultant $\mathbf{R}$ onto the $b$-axis.

$$
\begin{aligned}
& R_{1}=100 \mathrm{~N}=(80)^{2}+(100)^{2}-2(80)(100) \cos 130^{\circ} \quad R=163.4 \mathrm{~N} \\
& F_{b}=80+100 \cos 50^{\circ}=144.3 \mathrm{~N}
\end{aligned}
$$

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### 2.4 Moments

$\square$ A force can also tend to rotate a body about an axis
$\square$ Moment is also referred to as torque
$\square$ The magnitude of this tendency depends on:

* Magnitude $F$ of the force
$\%$ Effective length $d$ of the wrench handle


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### 2.5 Moments

$\square$ Moment about a Point

$$
M=F d
$$

* Plus sign for counterclockwise moments
- Minus sign for clockwise moments
: Sign consistency within a given problem is essential.

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### 2.5 Moments

- The Cross Product

$$
\mathbf{M}=\mathbf{r} \times \mathbf{F}
$$

*The moment of $\mathbf{F}$ about point $A$
$\% \mathbf{r}$ is a position vector which runs from the moment reference point $A$ to any point on the line of action of $\mathbf{F}$
\% We must maintain the sequence $\mathbf{r} \times \mathbf{F}$, because the sequence $\mathbf{F} \times \mathbf{r}$ would produce a vector with a sense opposite to that of the correct moment.


$$
M=F r \sin \alpha=F d
$$

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### 2.5 Moments

$\square$ Varignon's Theorem
The moment of a force about any point is equal to the sum of the moments of the components of the force about the same point.
$\mathbf{M}_{O}=\mathbf{r} \times \mathbf{R}$
$\mathbf{R}=\mathbf{P}+\mathbf{Q}$
$\longrightarrow \mathbf{r} \times \mathbf{R}=\mathbf{r} \times(\mathbf{P}+\mathbf{Q})$
$\longrightarrow \quad \mathbf{M}_{O}=\mathbf{r} \times \mathbf{R}=\mathbf{r} \times \mathbf{P}+\mathbf{r} \times \mathbf{Q}$

$\longrightarrow M_{O}=R d=-p P+q Q$
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## Sample Problem 2/5

Calculate the magnitude of the moment about the base point $O$ of the $600-\mathrm{N}$ force in five different ways.

Solution. (I)

$$
\begin{aligned}
& d=4 \cos 40^{\circ}+2 \sin 40^{\circ}=4.35 \mathrm{~m} \\
& M=F d \\
& M_{O}=600(4.35)=2610 \mathrm{~N} \cdot \mathrm{~m}
\end{aligned}
$$



Solution. (II) $F_{1}=600 \cos 40^{\circ}=460 \mathrm{~N}, \quad F_{2}=600 \sin 40^{\circ}=386 \mathrm{~N}$

$$
M_{O}=460(4)+386(2)=2610 \mathrm{~N} \cdot \mathrm{~m}
$$



Solution. (III)

$$
\begin{aligned}
& d_{1}=4+2 \tan 40^{\circ}=5.68 \mathrm{~m} \\
& M_{O}=460(5.68)=2610 \mathrm{~N} \cdot \mathrm{~m}
\end{aligned}
$$

Solution. (IV)

$$
\begin{aligned}
& d_{2}=2+4 \cot 40^{\circ}=6.77 \mathrm{~m} \\
& M_{O}=386(6.77)=2610 \mathrm{~N} \cdot \mathrm{~m}
\end{aligned}
$$

Solution. (V)

$$
\begin{aligned}
\mathbf{M}_{O} & =\mathbf{r} \times \mathbf{F}=(2 \mathbf{i}+4 \mathbf{j}) \times 600\left(\mathbf{i} \cos 40^{\circ}-\mathbf{j} \sin 40^{\circ}\right) \\
& =-2610 \mathbf{k} \mathrm{~N} \cdot \mathrm{~m} \\
M_{O} & =2610 \mathrm{~N} \cdot \mathrm{~m}
\end{aligned}
$$

### 2.5 COUPLE

$\square$ The moment produced by two equal, opposite, and noncollinear forces is called a couple.
$\square$ The forces only effect is to produce a tendency of rotation

$$
M=F(a+d)-F a
$$

$$
\longrightarrow \quad M=F d
$$




Counterclockwise couple

Clockwise couple
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### 2.5 COUPLE

$\square$ Vector Algebra Method

$$
\begin{aligned}
& \mathbf{M}=\mathbf{r}_{A} \times \mathbf{F}+\mathbf{r}_{B} \times(-\mathbf{F})=\left(\mathbf{r}_{A}-\mathbf{r}_{B}\right) \times \mathbf{F} \\
& \longrightarrow \mathbf{M}=\mathbf{r} \times \mathbf{F}
\end{aligned}
$$

$\%$ The moment expression contains no reference to the moment center $O$ and, therefore, is the same for all moment centers. Thus, we may represent $\mathbf{M}$ by a free vector.

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### 2.5 COUPLE

## $\square$ Equivalent Couples

Changing the values of $F$ and $d$ does not change a given couple as long as the product $F d$ remains the same.

* Likewise, a couple is not affected if the forces act in a different but parallel plane.

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### 2.5 COUPLE

## $\square$ Force-Couple Systems

*The effect of a force acting on a body:
$\checkmark$ Push or pull the body in the direction of the force
$\checkmark$ Rotate the body about any fixed axis which does not intersect the line of the force


* By reversing this process, we can combine a given couple and a force which lies in the plane of the couple (normal to the couple vector) to produce a single, equivalent force.
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## Sample Problem 2/7

The rigid structural member is subjected to a couple consisting of the two $100-\mathrm{N}$ forces. Replace this couple by an equivalent couple consisting of the two forces $\mathbf{P}$ and $-\mathbf{P}$, each of which has a magnitude of 400 N . Determine the proper angle $\theta$.

$$
\begin{aligned}
{[M} & =F d] \\
M & =400(0.040) \cos \theta \\
10 & =(400)(0.040) \cos \theta \\
\theta & =\cos ^{-1} \frac{10}{16}=51.3^{\circ}
\end{aligned}
$$

$$
M=100(0.1)=10 \mathrm{~N} \cdot \mathrm{~m}
$$



## Sample Problem 2/8

Replace the horizontal 80 -lb force acting on the lever by an equivalent system consisting of a force at $O$ and a couple.

$$
[M=F d] \quad M=80\left(9 \sin 60^{\circ}\right)=624 \mathrm{lb}-\mathrm{in} .
$$


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### 2.6 RESULTANT

$\square$ The resultant of a system of forces:
The simplest force combination which can replace the original forces without altering the external effect on the rigid body to which the forces are applied
$\square$ Equilibrium of a body:
The condition in which the resultant of all forces acting on the body is zero.

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### 2.6 Resultant

$\square$ The resultant of a system of forces

$$
\begin{gathered}
\mathbf{R}=\mathbf{F}_{1}+\mathbf{F}_{2}+\mathbf{F}_{3}+\cdots=\Sigma \mathbf{F} \\
R_{x}=\Sigma F_{x} \quad R_{y}=\Sigma F_{y} \quad R=\sqrt{\left(\Sigma F_{x}\right)^{2}+\left(\Sigma F_{y}\right)^{2}} \\
\theta=\tan ^{-1} \frac{R_{y}}{R_{x}}=\tan ^{-1} \frac{\Sigma F_{y}}{\Sigma F_{x}}
\end{gathered}
$$

### 2.6 RESULTANT

$\square$ Algebraic Method



### 2.6 Resultant

Principle of Moments$$
\begin{gathered}
\mathbf{R}=\Sigma \mathbf{F} \\
M_{O}=\Sigma M=\Sigma(F d) \\
R d=M_{O}
\end{gathered}
$$

*This extends Varignon's theorem to the case of nonconcurrent force systems.

* The three forces have a zero resultant force but have a resultant clockwise couple ( $\mathrm{M}=\mathrm{F} 3 \mathrm{~d}$ )

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## Sample Problem 2/9

Determine the resultant of the four forces and one couple which act on the plate shown.

Solution. Point $O$ is selected as a convenient reference point


$$
\left[R d=\left|M_{O}\right|\right] \quad 148.3 d=237 \quad d=1.600 \mathrm{~m}
$$

$$
\begin{aligned}
& {\left[R_{x}=\Sigma F_{x}\right] \quad R_{x}=40+80 \cos 30^{\circ}-60 \cos 45^{\circ}=66.9 \mathrm{~N}} \\
& {\left[R_{y}=\Sigma F_{y}\right]} \\
& R_{y}=50+80 \sin 30^{\circ}+60 \cos 45^{\circ}=132.4 \mathrm{~N} \\
& {\left[R=\sqrt{R_{x}{ }^{2}+R_{y}{ }^{2}}\right] \quad R=\sqrt{(66.9)^{2}+(132.4)^{2}}=148.3 \mathrm{~N}} \\
& {\left[\theta=\left.\tan \right|^{-1} \frac{R_{y}}{R_{x}}\right] \quad \theta=\tan ^{-1} \frac{132.4}{66.9}=63.2^{\circ}} \\
& {\left[M_{O}=\Sigma(F d)\right] \quad M_{O}=140-50(5)+60 \cos 45^{\circ}(4)-60 \sin 45^{\circ}(7)} \\
& =-237 \mathrm{~N} \cdot \mathrm{~m}
\end{aligned}
$$

$R_{y} b=\left|M_{O}\right| \quad$ and $\quad b=\frac{237}{132.4}=1.792 \mathrm{~m}$

$$
\mathbf{r} \times \mathbf{R}=\mathbf{M}_{O}
$$

$$
\longrightarrow(x \mathbf{i}+y \mathbf{j}) \times(66.9 \mathbf{i}+132.4 \mathbf{j})=-237 \mathbf{k}
$$

$$
(132.4 x-66.9 y) \mathbf{k}=-237 \mathbf{k}
$$

$$
132.4 x-66.9 y=-237
$$

$\longrightarrow$ By setting $y=0$, we obtain $x=-1.792 \mathrm{~m}$


### 2.7 3D Force Systems: Rectangular Components

$\square$ It is often necessary to resolve a force into its three mutually perpendicular components

$$
\begin{array}{ll}
F_{x}=F \cos \theta_{x} & F=\sqrt{F_{x}{ }^{2}+F_{y}{ }^{2}+F_{z}{ }^{2}} \\
F_{y}=F \cos \theta_{y} & \mathbf{F}=F_{x} \mathbf{i}+F_{y} \mathbf{j}+F_{z} \mathbf{k} \\
F_{z}=F \cos \theta_{z} & \mathbf{F}=F\left(\mathbf{i} \cos \theta_{x}+\mathbf{j} \cos \theta_{y}+\mathbf{k} \cos \theta_{z}\right) \\
\hline
\end{array}
$$

$$
\mathbf{F}=F(l \mathbf{i}+m \mathbf{j}+n \mathbf{k})
$$

$$
\mathbf{F}=F \mathbf{n}_{F}
$$



### 2.7 3D Force Systems: Rectangular Components

- Direction of a force:
(a) Specification by two points on the line of action of the force.

$$
\mathbf{F}=F \mathbf{n}_{F}=F \frac{\stackrel{\rightharpoonup}{A B}}{\overrightarrow{A B}}=F \frac{\left(x_{2}-x_{1}\right) \mathbf{i}+\left(y_{2}-y_{1}\right) \mathbf{j}+\left(z_{2}-z_{1}\right) \mathbf{k}}{\sqrt{\left(x_{2}-x_{1}\right)^{2}+\left(y_{2}-y_{1}\right)^{2}+\left(z_{2}-z_{1}\right)^{2}}}
$$



### 2.7 3D Force Systems: Rectangular Components

$\square$ Direction of a force:

* (b) Specification by two angles which orient the line of action of the force.

$$
\begin{aligned}
F_{x y} & =F \cos \phi \\
F_{z} & =F \sin \phi
\end{aligned}
$$

$$
\longrightarrow \quad \begin{aligned}
& F_{x}=F_{x y} \cos \theta=F \cos \phi \cos \theta \\
& F_{y}=F_{x y} \sin \theta=F \cos \phi \sin \theta
\end{aligned}
$$


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### 2.7 3D Force Systems: Rectangular Components

## $\square$ Dot Product

Expressing the rectangular components of a force $\mathbf{F}$ (or any other vector) with the aid of the vector operation known as the dot or scalar product.

$$
\mathbf{P} \cdot \mathbf{Q}=P Q \cos \alpha
$$

$$
\begin{aligned}
& \mathbf{i} \cdot \mathbf{i}=\mathbf{j} \cdot \mathbf{j}=\mathbf{k} \cdot \mathbf{k}=1 \\
& \mathbf{i} \cdot \mathbf{j}=\mathbf{j} \cdot \mathbf{i}=\mathbf{i} \cdot \mathbf{k}=\mathbf{k} \cdot \mathbf{i}=\mathbf{j} \cdot \mathbf{k}=\mathbf{k} \cdot \mathbf{j}=0
\end{aligned}
$$



$$
\begin{aligned}
F_{n}=\mathbf{F} \cdot \mathbf{n} & =F(l \mathbf{i}+m \mathbf{j}+n \mathbf{k}) \cdot(\alpha \mathbf{i}+\beta \mathbf{j}+\gamma \mathbf{k}) \\
& =F(l \alpha+m \beta+n \gamma)
\end{aligned}
$$

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### 2.7 3D Force Systems: Rectangular Components

- Dot Product

$$
\mathbf{n}=\alpha \mathbf{i}+\beta \mathbf{j}+\gamma \mathbf{k}
$$

$\longrightarrow F_{n}=\mathbf{F} \cdot \mathbf{n}=F(l \mathbf{i}+m \mathbf{j}+n \mathbf{k}) \cdot(\alpha \mathbf{i}+\beta \mathbf{j}+\gamma \mathbf{k})$

$$
=F(l \alpha+m \beta+n \gamma)
$$



### 2.7 3D Force Systems: Rectangular Components

$\square$ Angle between Two Vectors

$$
\begin{aligned}
& \theta=\cos ^{-1} \frac{\mathbf{P} \cdot \mathbf{Q}}{P Q} \\
& \theta=\cos ^{-1} \frac{\mathbf{F} \cdot \mathbf{n}}{F}
\end{aligned}
$$



## Sample Problem 2/10

A force $\mathbf{F}$ with a magnitude of 100 N is applied at the origin $O$ of the axes $x-y-z$ as shown. The line of action of $\mathbf{F}$ passes through a point $A$ whose coordinates are $3 \mathrm{~m}, 4 \mathrm{~m}$, and 5 m . Determine ( $a$ ) the $x, y$, and $z$ scalar components of $\mathbf{F}$, (b) the projection $F_{x y}$ of $\mathbf{F}$ on the $x-y$ plane, and (c) the projection $F_{O B}$ of $\mathbf{F}$ along the line $O B$.

$$
\begin{aligned}
\mathbf{F} & =F \mathbf{n}_{O A}=F \frac{\overrightarrow{O A}}{\overrightarrow{O A}}=100\left[\frac{3 \mathbf{i}+4 \mathbf{j}+5 \mathbf{k}}{\sqrt{3^{2}+4^{2}+5^{2}}}\right] \\
& =100[0.424 \mathbf{i}+0.566 \mathbf{j}+0.707 \mathbf{k}] \\
& =42.4 \mathbf{i}+56.6 \mathbf{j}+70.7 \mathbf{k} \mathrm{~N}
\end{aligned}
$$

$$
\longrightarrow \quad F_{x}=42.4 \mathrm{~N} \quad F_{y}=56.6 \mathrm{~N} \quad F_{z}=70.7 \mathrm{~N}
$$

$$
\cos \theta_{x y}=\frac{\sqrt{3^{2}+4^{2}}}{\sqrt{3^{2}+4^{2}+5^{2}}}=0.707
$$

$$
\longrightarrow F_{x y}=F \cos \theta_{x y}=100(0.707)=70.7 \mathrm{~N}
$$



$$
\mathbf{n}_{O B}=\frac{\overrightarrow{O B}}{\overline{O B}}=\frac{6 \mathbf{i}+6 \mathbf{j}+2 \mathbf{k}}{\sqrt{6^{2}+6^{2}+2^{2}}}=0.688 \mathbf{i}+0.688 \mathbf{j}+0.229 \mathbf{k}
$$

$$
F_{O B}=\mathbf{F} \cdot \mathbf{n}_{O B}=(42.4 \mathbf{i}+56.6 \mathbf{j}+70.7 \mathbf{k}) \cdot(0.688 \mathbf{i}+0.688 \mathbf{j}+0.229 \mathbf{k})
$$

$$
=(42.4)(0.688)+(56.6)(0.688)+(70.7)(0.229)
$$

$$
=84.4 \mathrm{~N}
$$

$$
\begin{aligned}
\longrightarrow \mathbf{F}_{O B} & =\mathbf{F} \cdot \mathbf{n}_{O B} \mathbf{n}_{O B} \\
& =84.4(0.688 \mathbf{i}+0.688 \mathbf{j}+0.229 \mathbf{k}) \\
& =58.1 \mathbf{i}+58.1 \mathbf{j}+19.35 \mathbf{k} \mathrm{~N}
\end{aligned}
$$



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### 2.8 3D Force Systems: Moments and Couple

- Moments in Three Dimensions

$$
\mathbf{M}_{O}=\mathbf{r} \times \mathbf{F}
$$



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### 2.8 3D Force Systems: Moments and Couple

$\square$ Evaluating the Cross Product

$$
\mathbf{M}_{O}=\left|\begin{array}{lll}
\mathbf{i} & \mathbf{j} & \mathbf{k} \\
r_{x} & r_{y} & r_{z} \\
F_{x} & F_{y} & F_{z}
\end{array}\right|
$$

$$
\mathbf{M}_{O}=\left(r_{y} F_{z}-r_{z} F_{y}\right) \mathbf{i}+\left(r_{z} F_{x}-r_{x} F_{z}\right) \mathbf{j}+\left(r_{x} F_{y}-r_{y} F_{x}\right) \mathbf{k}
$$

$$
M_{x}=r_{y} F_{z}-r_{z} F_{y} \quad M_{y}=r_{z} F_{x}-r_{x} F_{z} \quad M_{z}=r_{x} F_{y}-r_{y} F_{x}
$$



### 2.8 3D Force Systems: Moments and Couple

$\square$ Moment about an Arbitrary Axis

$$
\mathbf{M}_{\lambda}=(\mathbf{r} \times \mathbf{F} \cdot \mathbf{n}) \mathbf{n}
$$

$$
\left|\mathbf{M}_{\lambda}\right|=M_{\lambda}=\left|\begin{array}{lll}
r_{x} & r_{y} & r_{z} \\
F_{x} & F_{y} & F_{z} \\
\alpha & \beta & \gamma
\end{array}\right|
$$



### 2.8 3D Force Systems: Moments and Couple

$\square$ Varignon's Theorem in Three Dimensions:
For a system of concurrent forces $\mathbf{F}_{1}, \mathbf{F}_{2}, \mathbf{F}_{3}, \ldots$ The sum of the moments about $O$

$$
\begin{aligned}
\mathbf{r} \times \mathbf{F}_{1}+\mathbf{r} \times \mathbf{F}_{2}+\mathbf{r} \times \mathbf{F}_{3}+\cdots & =\mathbf{r} \times\left(\mathbf{F}_{1}+\mathbf{F}_{2}+\mathbf{F}_{3}+\cdots\right) \\
& =\mathbf{r} \times \Sigma \mathbf{F}
\end{aligned}
$$

$$
\longrightarrow \mathbf{M}_{O}=\Sigma(\mathbf{r} \times \mathbf{F})=\mathbf{r} \times \mathbf{R}
$$



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### 2.8 3D Force Systems: Moments and Couple

$\square$ Couples in Three Dimensions:

$$
\mathbf{M}=\mathbf{r}_{A} \times \mathbf{F}+\mathbf{r}_{B} \times(-\mathbf{F})=\left(\mathbf{r}_{A}-\mathbf{r}_{B}\right) \times \mathbf{F}
$$

$$
\longrightarrow \mathbf{M}=\mathbf{r} \times \mathbf{F}
$$

*The moment of a couple is a free vector.

* The moment of a force about a point is a sliding vector.



### 2.8 3D Force Systems: Moments and Couple

Couple vectors obey all of the rules which govern vector quantities.

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## Sample Problem 2/11

Determine the moment of force $\mathbf{F}$ about point $O(a)$ by inspection and $(b)$ by the formal cross-product definition $\mathbf{M}_{O}=\mathbf{r} \times \mathbf{F}$.

$$
\mathbf{M}_{O}=-c F \mathbf{i}+a F \mathbf{k}=F(-c \mathbf{i}+a \mathbf{k})
$$

$$
\begin{aligned}
\mathbf{M}_{O}=\mathbf{r} \times \mathbf{F}=(a \mathbf{i}+c \mathbf{k}) \times F \mathbf{j} & =a F \mathbf{k}-c F \mathbf{i} \\
& =F(-c \mathbf{i}+a \mathbf{k})
\end{aligned}
$$



## Sample Problem 2/12

The turnbuckle is tightened until the tension in cable $A B$ is 2.4 kN . Determine the moment about point $O$ of the cable force acting on point $A$ and the magnitude of this moment.

$$
\begin{aligned}
\mathbf{T} & =T \mathbf{n}_{A B}=2.4\left[\frac{0.8 \mathbf{i}+1.5 \mathbf{j}-2 \mathbf{k}}{\sqrt{0.8^{2}+1.5^{2}+2^{2}}}\right] \\
& =0.731 \mathbf{i}+1.371 \mathbf{j}-1.829 \mathbf{k} \mathrm{kN}
\end{aligned}
$$

$$
\begin{aligned}
\mathbf{M}_{O} & =\mathbf{r}_{O A} \times \mathbf{T}=(1.6 \mathbf{i}+2 \mathbf{k}) \times(0.731 \mathbf{i}+1.371 \mathbf{j}-1.829 \mathbf{k}) \\
& =-2.74 \mathbf{i}+4.39 \mathbf{j}+2.19 \mathbf{k} \mathrm{kN} \cdot \mathrm{~m}
\end{aligned}
$$

$\longrightarrow M_{O}=\sqrt{2.74^{2}+4.39^{2}+2.19^{2}}=5.62 \mathrm{kN} \cdot \mathrm{m}$


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## Sample Problem 2/14

Determine the magnitude and direction of the couple $\mathbf{M}$ which will replace the two given couples and still produce the same external effect on the block. Specify the two forces $\mathbf{F}$ and $-\mathbf{F}$, applied in the two faces of the block parallel to the $y-z$ plane, which may replace the four given forces. The $30-\mathrm{N}$ forces act parallel to the $y-z$ plane.
$M_{1}=30(0.06)=1.80 \mathrm{~N} \cdot \mathrm{~m}$.
$M_{y}=1.80 \sin 60^{\circ}=1.559 \mathrm{~N} \cdot \mathrm{~m}$

$$
M_{z}=-2.50+1.80 \cos 60^{\circ}=-1.600 \mathrm{~N} \cdot \mathrm{~m}
$$

$\longrightarrow M=\sqrt{(1.559)^{2}+(-1.600)^{2}}=2.23 \mathrm{~N} \cdot \mathrm{~m}$
$\longrightarrow \theta=\tan ^{-1} \frac{1.559}{1.600}=\tan ^{-1} 0.974=44.3^{\circ}$

$$
[M=F d] \quad \longrightarrow \quad F=\frac{2.23}{0.10}=22.3 \mathrm{~N}
$$



### 2.9 3D Force Systems: Resultant

$$
\begin{gathered}
\mathbf{R}=\mathbf{F}_{1}+\mathbf{F}_{2}+\mathbf{F}_{3}+\cdots=\Sigma \mathbf{F} \\
\mathbf{M}=\mathbf{M}_{1}+\mathbf{M}_{2}+\mathbf{M}_{3}+\cdots=\Sigma(\mathbf{r} \times \mathbf{F})
\end{gathered}
$$



$$
\longrightarrow \begin{gathered}
R_{x}=\Sigma F_{x} \quad R_{y}=\Sigma F_{y} \quad R_{z}=\Sigma F_{z} \\
R=\sqrt{\left(\Sigma F_{x}\right)^{2}+\left(\Sigma F_{y}\right)^{2}+\left(\Sigma F_{z}\right)^{2}} \\
\mathbf{M}_{x}=\Sigma(\mathbf{r} \times \mathbf{F})_{x} \quad \mathbf{M}_{y}=\Sigma(\mathbf{r} \times \mathbf{F})_{y} \quad \mathbf{M}_{z}=\Sigma(\mathbf{r} \times \mathbf{F})_{z} \\
M=\sqrt{M_{x}^{2}+M_{y}^{2}+M_{z}^{2}}
\end{gathered}
$$

## Sample Problem 2/16

Determine the resultant of the force and couple system which acts on the rectangular solid.

$$
\mathbf{R}=\Sigma \mathbf{F}=(80-80) \mathbf{i}+(100-100) \mathbf{j}+(50-50) \mathbf{k}=\mathbf{0} \mathrm{lb}
$$

$\mathbf{M}_{O}=[50(16)-700] \mathbf{i}+[80(12)-960] \mathbf{j}+[100(10)-1000] \mathbf{k}$ lb-in. $=100 \mathrm{ilb}-\mathrm{in}$.


## Sample Problem 2/17

Determine the resultant of the system of parallel forces which act on the plate. Solve with a vector approach.

$$
\begin{aligned}
\mathbf{R} & =\Sigma \mathbf{F}=(200+500-300-50) \mathbf{j}=350 \mathbf{j} \mathbf{N} \\
\mathbf{M}_{O} & =[50(0.35)-300(0.35)] \mathbf{i}+[-50(0.50)-200(0.50)] \mathbf{k} \\
& =-87.5 \mathbf{i}-125 \mathbf{k} \mathrm{~N} \cdot \mathrm{~m}
\end{aligned}
$$

$$
\begin{aligned}
\mathbf{r} \times \mathbf{R} & =\mathbf{M}_{O} \\
(x \mathbf{i}+y \mathbf{j}+z \mathbf{k}) \times 350 \mathbf{j} & =-87.5 \mathbf{i}-125 \mathbf{k} \\
350 x \mathbf{k}-350 z \mathbf{i} & =-87.5 \mathbf{i}-125 \mathbf{k}
\end{aligned}
$$

$x=-0.357 \mathrm{~m}$ and $z=0.250 \mathrm{~m}$


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